Linearization of Nonlinear Models

- Most chemical process models are nonlinear, but they are often linearized to perform a simulation and stability analysis.
- Linear models are easier to understand (than nonlinear models) and are necessary for most control system design methods.

Single Variable Example

A general single variable nonlinear model

$$\frac{dx}{dt} = f(x)$$

 The function f(x) can be approximated by a Taylor series approximation around the steady-state operating point (x_s)

$$f(x) = f(x_s) + \frac{\partial f}{\partial x}\Big|_{x_s} \left(x - x_s\right) + \frac{1}{2} \frac{\partial^2 f}{\partial x^2}\Big|_{x_s} \left(x - x_s\right)^2 + \text{high order terms}$$

Neglect the quadratic and higher order terms

$$f(x) \approx f(x_s) + \frac{\partial f}{\partial x}\Big|_{x_s} (x - x_s)$$
The p

At steady-state

$$\frac{dx_s}{dt} = f(x_s) = 0$$

The partial derivative of f(x) with respect to x, evaluated at the steady-state

$$\frac{dx}{dt} = f(x) \approx \frac{\partial f}{\partial x}\bigg|_{x_s} (x - x_s)$$

• Since the derivative of a constant (x_s) is zero

$$\frac{dx}{dt} = \frac{d(x - x_s)}{dt}$$

$$\frac{d(x - x_s)}{dt} \approx \frac{\partial f}{\partial x}\Big|_{x} (x - x_s)$$

 We are often interested in deviations in a state from a steadystate operating point (deviation variable)

$$\left. \frac{d\overline{x}}{dt} \approx \frac{\partial f}{\partial x} \right|_{x_s} \overline{x}$$

 $\overline{x} = x - x_s$: the change or perturbation from a steady-state value

Write in state-space form

$$\frac{d\overline{x}}{dt} \approx a \, \overline{x}$$
 where $a = \frac{\partial f}{\partial x}\Big|_{x_s}$

One State Variable and One Input Variable

Consider a function with one state variable and one input variable

$$\dot{x} = \frac{dx}{dt} = f(x, u)$$

Using a Taylor Series Expansion for f(x,u)

$$\dot{x} = f(x_s, u_s) + \frac{\partial f}{\partial x} \Big|_{x_s, u_s} (x - x_s) + \frac{\partial f}{\partial u} \Big|_{x_s, u_s} (u - u_s) + \frac{1}{2} \frac{\partial^2 f}{\partial x^2} \Big|_{x_s, u_s} (x - x_s)^2$$

$$+ \frac{\partial^2 f}{\partial x \partial u} \Big|_{x_s, u_s} (x - x_s) (u - u_s) + \frac{1}{2} \frac{\partial^2 f}{\partial u^2} \Big|_{x_s, u_s} (u - u_s)^2 + \text{high order terms}$$

Truncating after the linear terms

zero
$$\frac{\dot{x} \approx f(x_s, u_s) + \frac{\partial f}{\partial x}\Big|_{x_s, u_s} (x - x_s) + \frac{\partial f}{\partial u}\Big|_{x_s, u_s} (u - u_s)}{\frac{d(x - x_s)}{dt} \approx \frac{\partial f}{\partial x}\Big|_{x_s, u_s} (x - x_s) + \frac{\partial f}{\partial u}\Big|_{x_s, u_s} (u - u_s)}$$

• Using deviation variables, $\overline{x} = x - x_s$ and $\overline{u} = u - u_s$

$$\left. \frac{d\overline{x}}{dt} \approx \frac{\partial f}{\partial x} \right|_{x_s, u_s} \overline{x} + \left. \frac{\partial f}{\partial u} \right|_{x_s, u_s} \overline{u}$$

Write in state-space form

$$\frac{d\overline{x}}{dt} \approx a \, \overline{x} + b \, \overline{u}$$
 where $a = \frac{\partial f}{\partial x}\Big|_{x_s, u_s}$ $b = \frac{\partial f}{\partial u}\Big|_{x_s, u_s}$

• If there is a single output that is a function of the state and input

$$y = g(x, u)$$

Perform a Taylor series expansion and truncate high order terms

$$g(x,u) \approx g(x_s, u_s) + \frac{\partial g}{\partial x}\Big|_{x_s, u_s} (x - x_s) + \frac{\partial g}{\partial u}\Big|_{x_s, u_s} (u - u_s) \qquad g(x_s, u_s) = y_s$$

$$y - y_s = \frac{\partial g}{\partial x}\Big|_{x_s, u_s} (x - x_s) + \frac{\partial g}{\partial u}\Big|_{x_s, u_s} (u - u_s)$$

$$\overline{y} = c \,\overline{x} + d \,\overline{u} \qquad \text{where} \quad c = \frac{\partial g}{\partial x}\Big|_{x_s, u_s} \qquad d = \frac{\partial g}{\partial u}\Big|_{x_s, u_s}$$

Linearization of Multistate Models

Two-state system

$$\dot{x}_1 = \frac{dx_1}{dt} = f_1(x_1, x_2, u)$$

$$\dot{x}_2 = \frac{dx_2}{dt} = f_2(x_1, x_2, u)$$

$$y = g(x_1, x_2, u)$$

 Perform Taylor series expansion of the nonlinear functions and neglect high-order terms

$$f_1(x_1, x_2, u) = f_1(x_{1s}, x_{2s}, u_s) + \frac{\partial f_1}{\partial x_1} \bigg|_{x_{1s}, x_{2s}, u_s} \left(x_1 - x_{1s} \right) + \frac{\partial f_1}{\partial x_2} \bigg|_{x_{1s}, x_{2s}, u_s} \left(x_2 - x_{2s} \right) + \frac{\partial f_1}{\partial u} \bigg|_{x_{1s}, x_{2s}, u_s} \left(u - u_s \right)$$

$$f_{2}(x_{1}, x_{2}, u) = f_{2}(x_{1s}, x_{2s}, u_{s}) + \frac{\partial f_{2}}{\partial x_{1}}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(x_{1} - x_{1s}\right) + \frac{\partial f_{2}}{\partial x_{2}}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(x_{2} - x_{2s}\right) + \frac{\partial f_{2}}{\partial u}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(u - u_{s}\right)$$

$$g(x_{1}, x_{2}, u) = g(x_{1s}, x_{2s}, u_{s}) + \frac{\partial g}{\partial x_{1}}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(x_{1} - x_{1s}\right) + \frac{\partial g}{\partial x_{2}}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(x_{2} - x_{2s}\right) + \frac{\partial g}{\partial u}\bigg|_{x_{1s}, x_{2s}, u_{s}} \left(u - u_{s}\right)$$

For the linearization about the steady-state

$$f_1(x_{1s}, x_{2s}, u_s) = f_2(x_{1s}, x_{2s}, u_s) = 0 g(x_{1s}, x_{2s}, u_s) = y_s$$

$$\frac{dx_1}{dt} = \frac{d(x_1 - x_{1s})}{dt} \frac{dx_2}{dt} = \frac{d(x_2 - x_{2s})}{dt}$$

We can write the state-space model

$$\begin{bmatrix} \frac{d(x_{1}-x_{1s})}{dt} \\ \frac{d(x_{2}-x_{2s})}{dt} \end{bmatrix} = \begin{bmatrix} \frac{\partial f_{1}}{\partial x_{1}} \Big|_{x_{1s},x_{2s},u_{s}} & \frac{\partial f_{1}}{\partial x_{2}} \Big|_{x_{1s},x_{2s},u_{s}} \end{bmatrix} \begin{bmatrix} x_{1}-x_{1s} \\ x_{2}-x_{2s} \end{bmatrix} + \begin{bmatrix} \frac{\partial f_{1}}{\partial u} \Big|_{x_{1s},x_{2s},u_{s}} \\ \frac{\partial f_{2}}{\partial u} \Big|_{x_{1s},x_{2s},u_{s}} \end{bmatrix} [u-u_{s}]$$

$$y-y_{s} = \begin{bmatrix} \frac{\partial g}{\partial x_{1}} \Big|_{x_{1s},x_{2s},u_{s}} & \frac{\partial g}{\partial x_{2}} \Big|_{x_{1s},x_{2s},u_{s}} \end{bmatrix} \begin{bmatrix} x_{1}-x_{1s} \\ x_{2}-x_{2s} \end{bmatrix} + \begin{bmatrix} \frac{\partial g}{\partial u} \Big|_{x_{1s},x_{2s},u_{s}} \end{bmatrix} [u-u_{s}]$$

$$\dot{\overline{x}} = \mathbf{A} \, \overline{\mathbf{x}} + \mathbf{B} \, \overline{\mathbf{u}}$$

$$\bar{\mathbf{y}} = \mathbf{C} \, \overline{\mathbf{x}} + \mathbf{D} \, \overline{\mathbf{u}}$$

Generalization

 Consider a general nonlinear model with n state variables, m input variables, and r output variables

$$\dot{x}_1 = f_1(x_1, \dots, x_n, u_1, \dots, u_m)$$

$$\vdots$$

$$\dot{x}_n = f_n(x_1, \dots, x_n, u_1, \dots, u_m)$$

$$y_1 = g_1(x_1, \dots, x_n, u_1, \dots, u_m)$$

$$\vdots$$

$$y_r = g_r(x_1, \dots, x_n, u_1, \dots, u_m)$$

$$\vdots$$

$$y_r = g_r(x_1, \dots, x_n, u_1, \dots, u_m)$$

$$\forall \mathbf{y} = \mathbf{g}(\mathbf{x}, \mathbf{u})$$

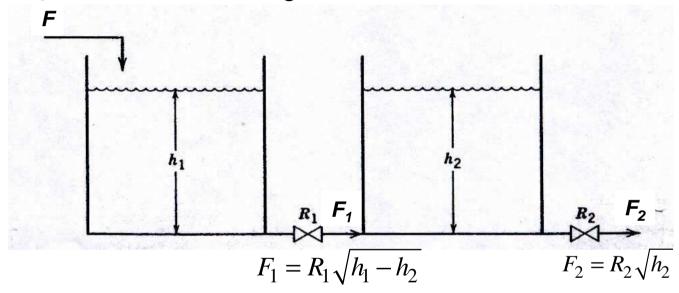
Elements of the linearization matrices

$$A_{ij} = \frac{\partial f_i}{\partial x_j} \bigg|_{\mathbf{x}_s, \mathbf{u}_s} \qquad B_{ij} = \frac{\partial f_i}{\partial u_j} \bigg|_{\mathbf{x}_s, \mathbf{u}_s} \qquad \mathbf{\bar{x}} = \mathbf{A} \, \mathbf{\bar{x}} + \mathbf{B} \, \mathbf{\bar{u}}$$

$$\mathbf{\bar{y}} = \mathbf{C} \, \mathbf{\bar{x}} + \mathbf{D} \, \mathbf{\bar{u}}$$
or
$$\mathbf{c}_{ij} = \frac{\partial g_i}{\partial x_j} \bigg|_{\mathbf{x}_s, \mathbf{u}_s} \qquad D_{ij} = \frac{\partial g_i}{\partial u_j} \bigg|_{\mathbf{x}_s, \mathbf{u}_s} \qquad \mathbf{\bar{x}} = \mathbf{A} \, \mathbf{x} + \mathbf{B} \, \mathbf{u} \qquad \text{(The "overbar" is } \mathbf{y} = \mathbf{C} \, \mathbf{x} + \mathbf{D} \, \mathbf{u} \qquad \text{usually dropped)}$$

Example: Interacting Tanks

 Two interacting tank in series with outlet flowrate being function of the square root of tank height



Modeling equations

$$\frac{dh_1}{dt} = \frac{F}{A_1} - \frac{R_1}{A_1} \sqrt{h_1 - h_2} = f_1(h_1, h_2, F)$$

$$\frac{dh_2}{dt} = \frac{R_1}{A_2} \sqrt{h_1 - h_2} - \frac{R_2}{A_2} \sqrt{h_2} = f_2(h_1, h_2, F)$$

- Assume only the second tank height is measured. The output, in deviation variable form is $y = h_2 h_{2s}$
- There are two state variables, one input variable, one one output variable

$$\mathbf{h_s} = \begin{bmatrix} h_{1s} \\ h_{2s} \end{bmatrix} \qquad \mathbf{x} = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} h_1 - h_{1s} \\ h_2 - h_{2s} \end{bmatrix} \qquad u = F - F_s$$

The element of the A (Jacobian) and B matrices

$$A_{11} = \frac{\partial f_{1}}{\partial h_{1}}\Big|_{\mathbf{h}_{s},F_{s}} = -\frac{R_{1}}{2A_{1}\sqrt{h_{1s} - h_{2s}}}$$

$$A_{12} = \frac{\partial f_{1}}{\partial h_{2}}\Big|_{\mathbf{h}_{s},F_{s}} = \frac{R_{1}}{2A_{1}\sqrt{h_{1s} - h_{2s}}}$$

$$B_{11} = \frac{\partial f_{1}}{\partial F}\Big|_{\mathbf{h}_{s},F_{s}} = \frac{1}{A_{1}}$$

$$A_{21} = \frac{\partial f_{2}}{\partial h_{1}}\Big|_{\mathbf{h}_{s},F_{s}} = \frac{R_{1}}{2A_{2}\sqrt{h_{1s} - h_{2s}}}$$

$$B_{21} = \frac{\partial f_{2}}{\partial F}\Big|_{\mathbf{h}_{s},F_{s}} = 0$$

$$A_{22} = \frac{\partial f_{2}}{\partial h_{2}}\Big|_{\mathbf{h}_{s},F_{s}} = -\frac{R_{1}}{2A_{2}\sqrt{h_{1s} - h_{2s}}} - \frac{R_{2}}{2A_{2}\sqrt{h_{2s}}}$$

Only the height of the second tank is measured

$$y = g(h_1, h_2, F) = h_2 - h_{2s}$$

$$C_{11} = \frac{\partial g}{\partial h_1} \Big|_{\mathbf{h}_s, F_s} = 0$$

$$C_{12} = \frac{\partial g_2}{\partial h_2} \Big|_{\mathbf{h}_s, F_s} = 1$$

The state-space model is

$$\begin{bmatrix} \frac{dx_1}{dt} \\ \frac{dx_2}{dt} \end{bmatrix} = \begin{bmatrix} -\frac{R_1}{2A_1\sqrt{h_{1s} - h_{2s}}} & \frac{R_1}{2A_1\sqrt{h_{1s} - h_{2s}}} \\ \frac{R_1}{2A_2\sqrt{h_{1s} - h_{2s}}} & -\frac{R_1}{2A_2\sqrt{h_{1s} - h_{2s}}} - \frac{R_2}{2A_2\sqrt{h_{2s}}} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} \frac{1}{A_1} \\ 0 \end{bmatrix} [u]$$

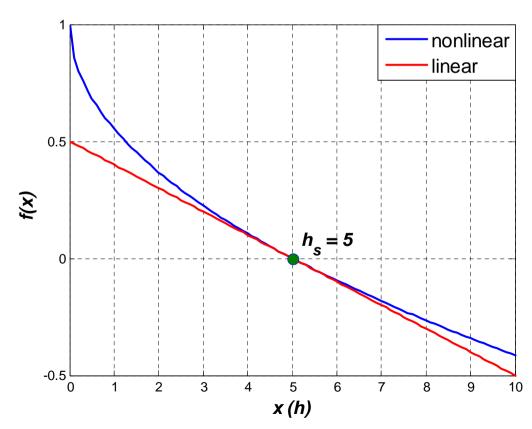
$$y = \begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \qquad (y = x_2 = h_2 - h_{2s})$$

Interpretation of Linearization

Consider the single tank problem (assume F is constant)

$$\frac{dh}{dt} = \frac{F}{A} - \frac{R}{A}\sqrt{h} = f(h, F) = 1 - \frac{1}{\sqrt{5}}\sqrt{h}$$

• Linearization $f(h,F) \approx 0 - \frac{1}{10}(h - h_s)$



The linear approximation works well between 3.5 to 7 feet

The two functions are exactly equal at the steady-state value of 5 feet

Exercise: interacting tanks

- Two interacting tank in series with outlet flowrate being function of the square root of tank height
 - Parameter values

$$R_1 = 2.5 \frac{\text{ft}^{2.5}}{\text{min}}$$
 $R_2 = \frac{5}{\sqrt{6}} \frac{\text{ft}^{2.5}}{\text{min}}$ $A_1 = 5 \text{ft}^2$ $A_2 = 10 \text{ft}^2$

- Input variable $F = 5 \text{ ft}^3/\text{min}$
- Steady-state height values : h_{1s} = 10, h_{2s} = 6
- Perform the following simulation using <u>state-space model</u>
 - What are the responses of tank height if the initial heights are $h_1(0)=12$ ft and $h_2(0)=7$ ft ?
 - Assume the system is at steady-state initially. What are the responses of tank height if
 - F changes from 5 to 7 ft³/min at t = 0
 - F has periodic oscillation of $F = 5 + \sin(0.2t)$
 - F changes from 5 to 4 ft³/min at t = 20

Stability of State-Space Models

- A state-space model is said to be stable if the response x(t) is bounded for all u(t) that is bounded
- Stability criterion for state-space model
 - The state-space model will exhibit a bounded response x(t) for all bounded u(t), if and only if all of the eigenvalues of A have negative real parts

(the stability is independent matrices **B** and **C**)

• Single variable equation $\dot{x} = a x$ has the solution

$$x(t) = e^{at}x(0) \implies \text{ stable if } a < 0$$

- The solution of $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x}$ is $\mathbf{x}(t) = e^{\mathbf{A}t}\mathbf{x}(0)$
 - Stable if all of the eigenvalues of A are less than zero
 - The response x(t) is oscillatory if the eigenvalues are complex

Exercise

Consider the following system equations

$$\dot{x}_1 = -0.5x_1 + x_2 \\ \dot{x}_2 = -2x_2$$

 $\dot{x}_2 = -2x_2$ - Find the responses of $\mathbf{x}(t)$ for $\mathbf{x}(0) = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$ and $\mathbf{x}(0) = \begin{bmatrix} -0.5547 \\ 0.8321 \end{bmatrix}$

(slow subspace v.s. fast subspace)

Consider the following system equations

$$\dot{x}_1 = 2x_1 + x_2$$

$$\dot{x}_2 = 2x_1 - x_2$$

- Find the responses of
$$\mathbf{x}(t)$$
 for $\mathbf{x}(0) = \begin{bmatrix} 0.2703 \\ -0.9628 \end{bmatrix}$ and $\mathbf{x}(0) = \begin{bmatrix} 0.8719 \\ 0.4896 \end{bmatrix}$

(stable subspace v.s. unstable subspace)

Note: Find eigenvalue and eigenvector of A \gg [V, D] = eig(A)